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LAMPIRAN

Lampiran 1. Dokumentasi Lapangan

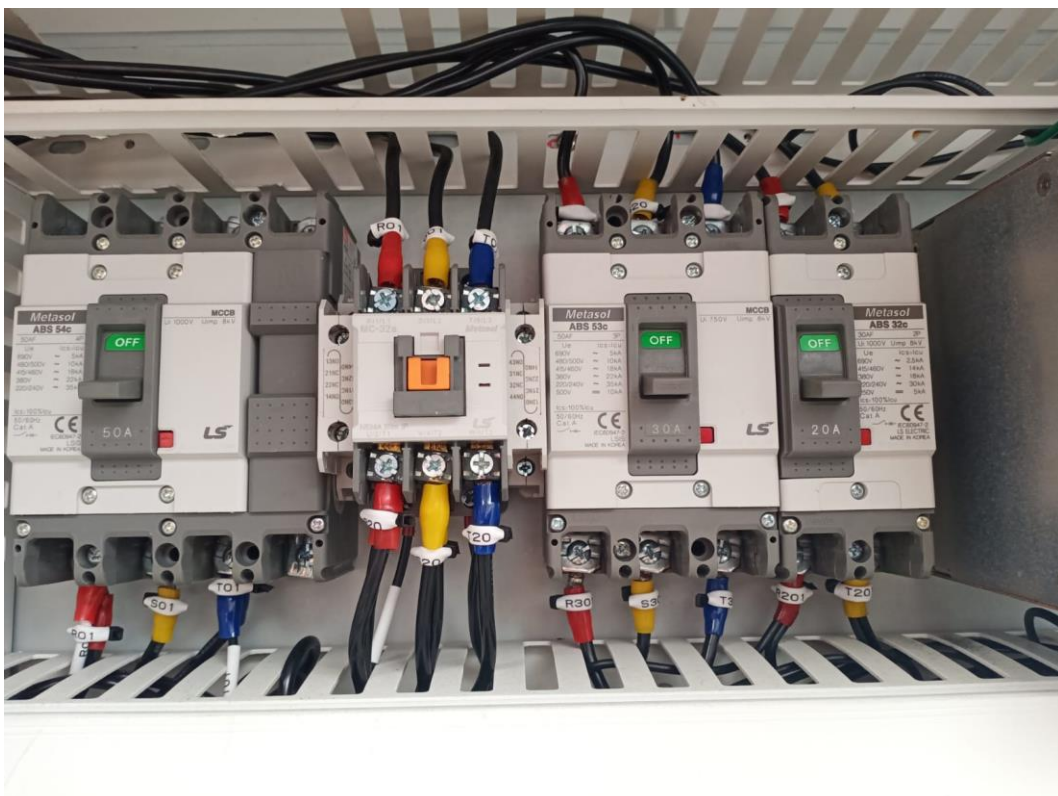








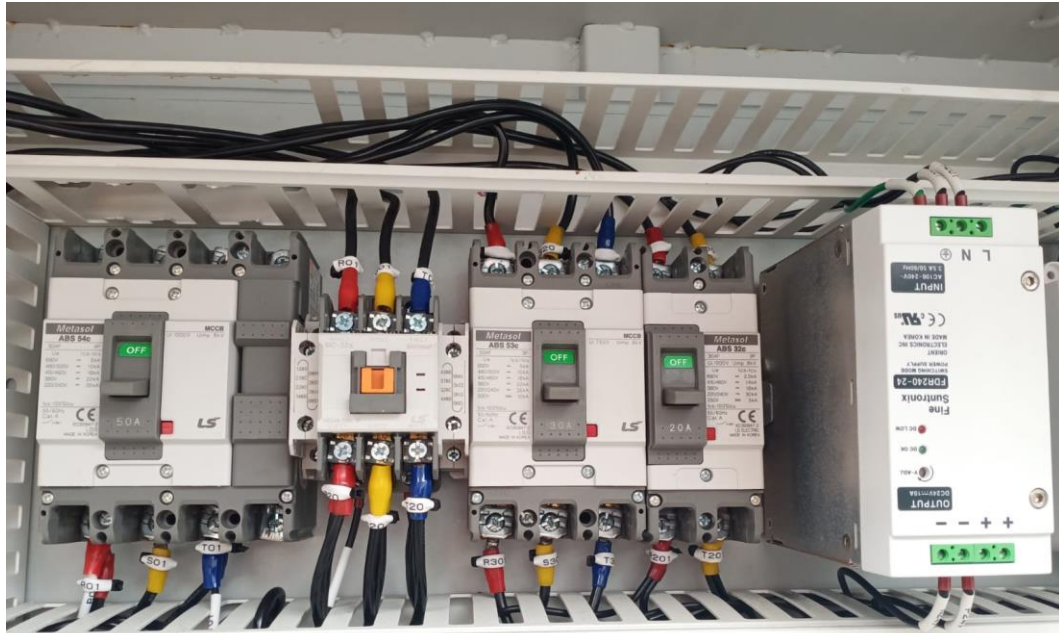














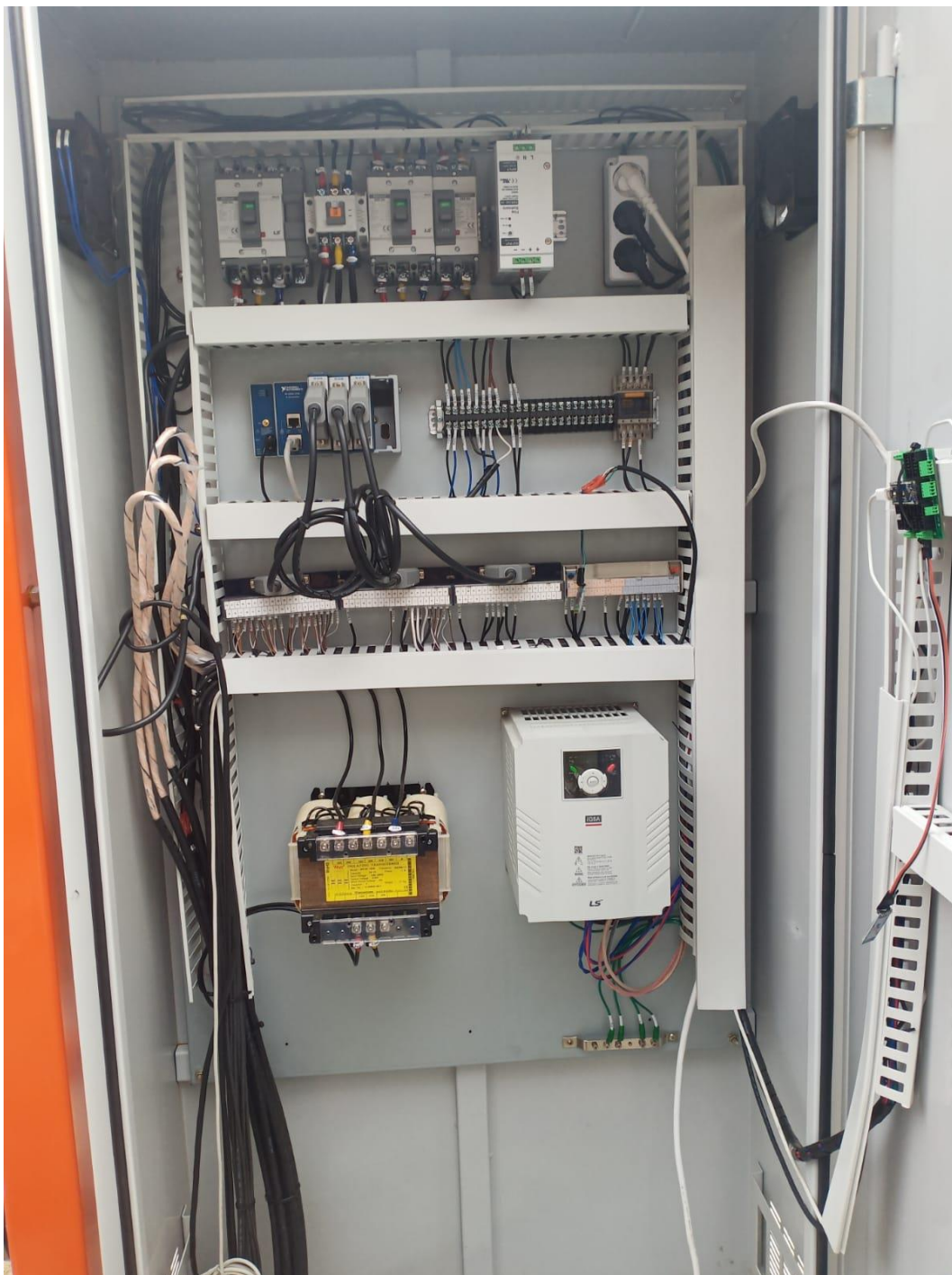


















**PEMERINTAH KABUPATEN LUWU UTARA
KECAMATAN MASAMBA
DESA PINCARA**

Alamat: Jln Poros Masamba-Rampi, Kode Pos 92961

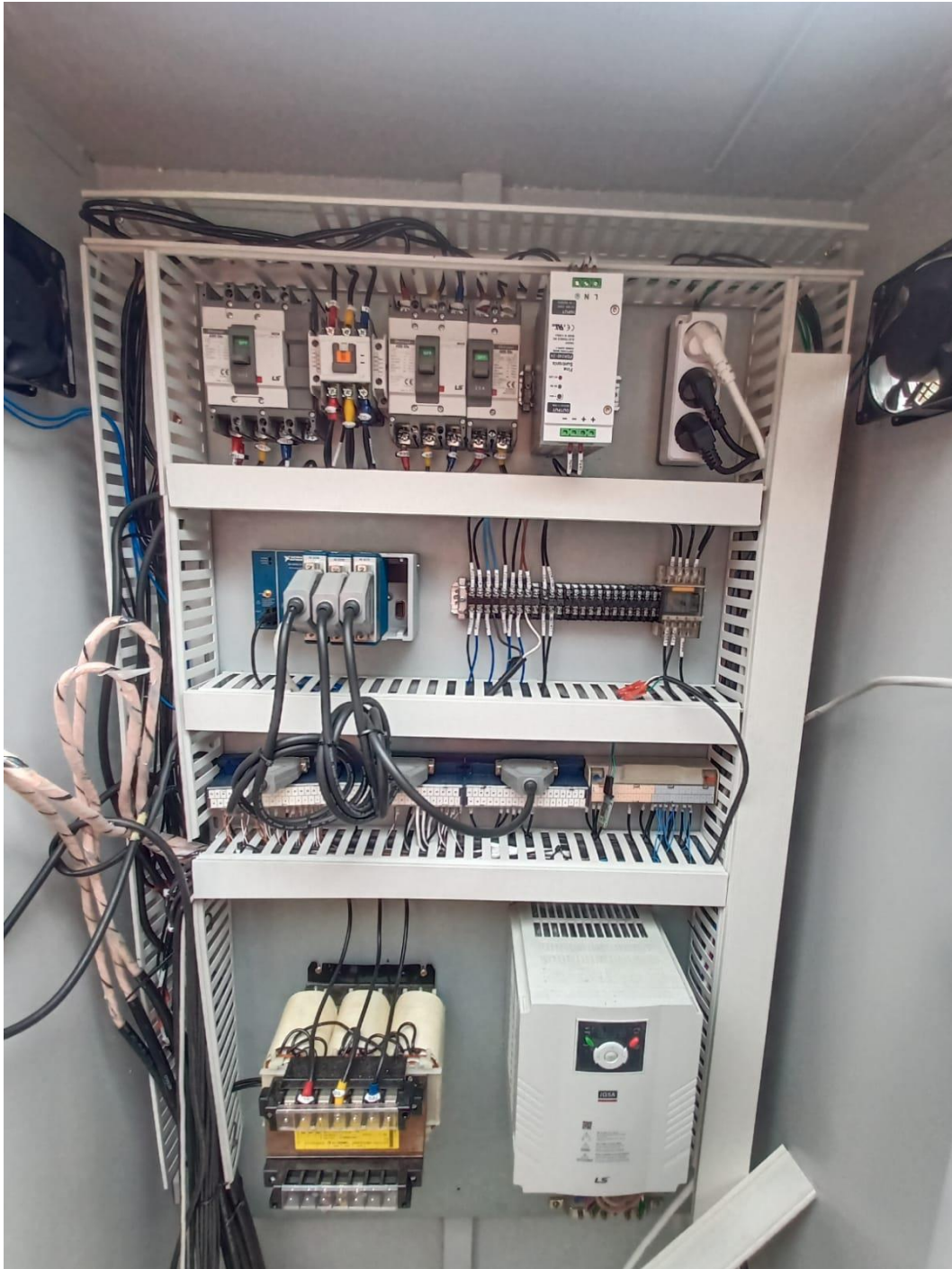
DAFTAR HADIR

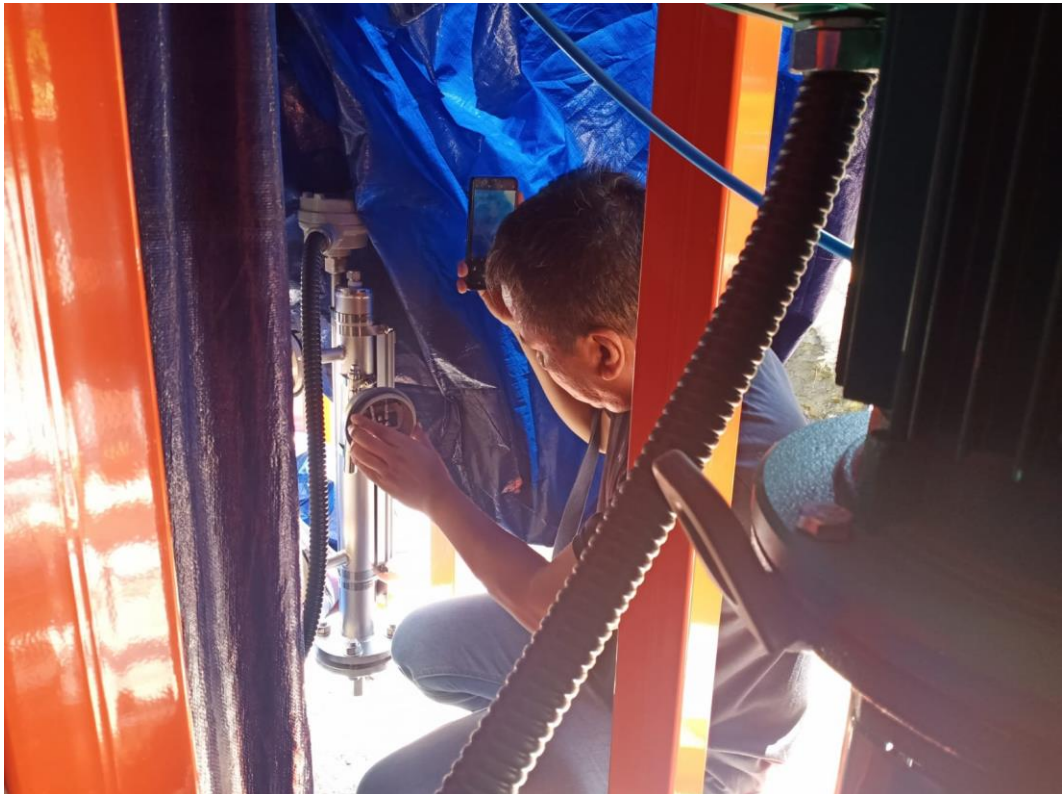
Hari/ Tanggal :
Tempat :
Kegiatan :

NO	NAMA	JABATAN	ALAMAT	NO HP	TANDA TANGAN
1					1
2					2
3	SABARUDDIN	Kadus	Pincara		3
4	WAWAR	BPD	S. Bomban		4
5	MUHLIS	APRAT	DS Pincara		5
6	NASRENS	MSI	S. Bomban		6
7	YAHYA Hidayah	MASYARAKAT	DS. Pincara		7
8	KURNIAWAN	MASYARAKAT	Desa Pincara		8
9	Muh. Rofi	Sekdes	—		9
10	Aly fadima	Mahasiswa	Desa Pincara		10
11	Nur Ain	Operator	DS. Salubanten		11
12	ROSTIKA	Kaur kaw.	Pincara		12
13	ISMAN	Masyarakat	Pincara		13
14	Diapor Arbie	Masy.	Pincara		14
15	RUSY	MASY.	PINCARA		15
16	Kacang	—	S. Bomban		16
17	MAHMUDIN	—	Pincara		17
18	MUHTAR	—	—		18
19					19
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Lampiran 2. Kode pemrograman matlab r2023a dalam analisis termal dan parameter simulasi

```
% Thermodynamic Properties Table R245fa
R245faTables = twoPhaseFluidTables([180,493],[0.05,4],50,50,100,...
'R245fa','C:\Program Files\REFPROP');

% Thermal Analysis

% Input
RP = py.ctREFPROP.ctREFPROP.REFPROPFunctionLibrary('C:\Program
Files\REFPROP');
Unit = RP.GETENUMdll(int8(0),'MASS BASE SI').iEnum; % Satuan Unit SI
Fluids = 'R245fa';
iMass = int8(1); % 0: molar fractions; 1: mass fractions
iFlag = int8(0); % 0: don't call SATSPLN; 1: call SATSPLN
z = {1.0}; % Cairan Murni, mole fraction = 1.0
T_Cndsr = Data_Temperature_Air_Sungai; % Kelvin
T_Evptr = Data_Temperature_Geothermal; % Kelvin
nIs_Trbn = 0.95; % Percent
nIs_Pump = 0.67; % Percent
P_Gas = RP.REFPROPdll(Fluids,'TQ','P',Unit,iMass,iFlag,T_Evptr,1,z);
P_Gas = double(P_Gas.Output);
P_Gas = P_Gas(1) % Pa
P_Work = 0.73;% Mpa

% Point 1 (Condensor Outlet - Pump Inlet)
P_Cndsr =
RP.REFPROPdll(Fluids,'TQ','P',Unit,iMass,iFlag,T_Cndsr,0,z);
P_Cndsr = double(P_Cndsr.Output);
P_Cndsr = P_Cndsr(1); % Pa
P1 = P_Cndsr;
h1 = RP.REFPROPdll(Fluids,'TP','H',Unit,iMass,iFlag,T_Cndsr,P1,z);
h1 = double(h1.Output);
h1 = h1(1); % J/Kg
s1 = RP.REFPROPdll(Fluids,'TH','S',Unit,iMass,iFlag,T_Cndsr,h1,z);
s1 = double(s1.Output);
s1 = s1(1); % J/Kg

% Point 2 (Pump Outlet - Evaporator Inlet)
P2 = P_Work * 1.0e6; % Pa
P_Evptr = P2; % Pa
s2_s = s1; % J/Kg
```

```

h2_s =
RP.REFPROPdll(Fluids,'PS','H',Unit,iMass,iFlag,P_Evptr,s2_s,z);
h2_s = double(h2_s.Output);
h2_s = h2_s(1); % J/Kg
h_1 = h1 * 0.001; % kJ/Kg
h_2s = h2_s * 0.001; % kJ/Kg
h_2 = h_1 + (h_2s - h_1) / nIs_Pump; % kJ/Kg

% Point 3 (Evaporator Outlet - Turbine inlet)
P3 = P_Evptr;
h3 =
RP.REFPROPdll(Fluids,'TPvap','H',Unit,iMass,iFlag,T_Evptr,P_Evptr,z)
;
h3 = double(h3.Output);
h3 = h3(1); % J/Kg
u3 =
RP.REFPROPdll(Fluids,'TPvap','E',Unit,iMass,iFlag,T_Evptr,P_Evptr,z)
;
u3 = double(u3.Output);
u3 = u3(1); % J/Kg
V3 =
RP.REFPROPdll(Fluids,'TPvap','v',Unit,iMass,iFlag,T_Evptr,P_Evptr,z)
;
V3 = double(V3.Output);
V3 = V3(1); % m^3/Kg
s3 =
RP.REFPROPdll(Fluids,'PT','S',Unit,iMass,iFlag,P_Evptr,T_Evptr,z);
s3 = double(s3.Output);
s3 = s3(1); % J/Kg

% Point 4 (Turbine Outlet - Condensor Inlet)
s4 = s3; % J/Kg
P4 = P1; % Pa
h4_s = RP.REFPROPdll(Fluids,'SP','H',Unit,iMass,iFlag,s4,P4,z);
h4_s = double(h4_s.Output);
h4_s = h4_s(1); % J/Kg
h_3 = h3 * 0.001; % kJ/Kg
h_4s = h4_s * 0.001;
h_4 = h_3 - nIs_Trbn * (h_3 - h_4s); % kJ/Kg
Turb_Isentropic = (h_3 - h_4)/(h_3 - h_4s ); % Percent

% Working Fluid Mass Flow Rate
m_dot = 0.0627777778;
h_g = RP.REFPROPdll(Fluids,'QT','H',Unit,iMass,iFlag,1,T_Evptr,z);
h_g = double(h_g.Output);
h_g = h_g(1); % J/Kg
h_gas = h_g * 0.001; % kJ/Kg

```

```

m_dotTemp = 31 / (h_gas - h_2); % Kg/s

% Evaporator Extracted Load
Q_Evptr = m_dotTemp * (h_3 - h_2);

% Condenser Rejected Load
Q_Cndsr = m_dotTemp * (h_4 - h_1);

% Turbine Power Output
W_Trbn = nIs_Trbn * m_dotTemp * (h_3 - h_4);

% Pump Consumed Power
W_Pump = m_dotTemp * (h_1 - h_2);

% Cycle Net Power Output
W_net = W_Trbn + W_Pump;

% Cycle Net Thermal Efficiency
n_Thermal = (W_net / Q_Evptr) * 100;

% PHE Geometry for Heat Transfer
Tank_Fluid_Volume = 0.2; % m^3
N_Plate = 80;
Gap_Plate = 0.004; % m
Length_Plate = 1; % m
Width_Plate = 0.4; % m
Thicknes_Plate = 0.01; % m
Internal_Surface_Roughnes = 3.6e-6; % m
Brass_Thermal_Conductivity = 125; % W/m*K
Cross_Sectional_Channel = Gap_Plate * Width_Plate; % m^2
Wetted_Perimeter = 2*(Gap_Plate + Width_Plate); % m
Cross_Section_HT = 2*(Gap_Plate + Width_Plate); % m
Fouling_Factor_TL = 0.0004; % K*m^2/W
Fouling_Factor_Rfgrnt_Vap = 0.0004; % K*m^2/W
Fouling_Factor_Rfgrnt_Liq = 0.0002; % K*m^2/W
Thermal_Resistance = (Thicknes_Plate / (Brass_Thermal_Conductivity *
Cross_Sectional_Channel)) / 1000; % K/kW
Delta_K_Subcooling = 0; % K
Delta_K_Superheat = 0; % K

```

% Output

```

disp('At state point 1 (Condenser Outlet - Pump Inlet):')
fprintf('P1 = %.3f MPa',P1 * 1.0e-6)
fprintf('T1 = %.1f °C',T_Cndsr - 273.15)
fprintf('H1 = %.2f kJ/kg',h_1)
fprintf('S1 = %.3f kJ/kg',s1 * 0.001)

disp('At state point 2 (Pump Outlet - Evaporator Inlet):')
fprintf('P2 = %.3f MPa',P_Evptr * 1.0e-6)
fprintf('T2 = %.1f °C',T_Evptr - 273.15)
fprintf('H2s = %.2f kJ/kg',h_2s)
fprintf('H2 = %.2f kJ/kg',h_2)
fprintf('H Gas = %.2f kJ/kg',h_gas)
fprintf('S2s = %.3f kJ/kg',s2_s * 0.001)

disp('At state point 3 (Evaporator Outlet - Turbine Inlet):')
fprintf('P3 = %.3f MPa',P3 * 1.0e-6)
fprintf('T3 = %.1f °C',T_Evptr - 273.15)
fprintf('H3 = %.2f kJ/kg',h_3)
fprintf('S3 = %.3f kJ/kg',s3 * 0.001)

disp('At state point 4 (Turbine Outlet - Condenser Inlet):')
fprintf('P4 = %.3f MPa',P4 * 1.0e-6)
fprintf('T4 = %.1f °C',T_Cndsr - 273.15)
fprintf('H4s = %.2f kJ/kg',h_4s)
fprintf('H4 = %.2f kJ/kg',h_4)
fprintf('S4 = %.3f kJ/kg',s4 * 0.001)

fprintf('Working Fluid Mass Flow Rate = %.4f kg/s',m_dotTemp)
fprintf('Evaporator Extracted Load = %.2f kW',Q_Evptr)
fprintf('Condenser Rejected Load = %.2f kW',Q_Cndsr)
fprintf('Turbine Power Output = %.2f kW',W_Trbn)
fprintf('Pump Consumed Power = %.2f kW',W_Pump)
fprintf('Cycle Net Power Output = %.2f kW',W_net)
fprintf('Thermal Efficiency = %.1f %',n_Thermal)

```

Lampiran 3. Data sheet pompa

Helix FIRST V 1606

wilo**Data sheet****Product data**

Product description	Helix FIRST V 1606
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Hydraulic data

Maximum inlet pressure p_{inf}	10 bar
Maximum operating pressure P_N	25 bar
Discharge port	DN 50
Min. fluid temperature T_{min}	-20 °C
Max. fluid temperature T_{max}	120 °C
Min. ambient temperature T_{min}	-15 °C
Max. ambient temperature T_{max}	50 °C

Motor data

Voltage tolerance	±10 %
Rated power P_2	7.5 kW
Rated current I_N	13.8 A
Rated speed n	2923 1/min
Power factor $\cos \varphi_{100}$	0.87
Motor efficiency 50% $\eta_{M 50\%}$	89.6 %
Motor efficiency 75% $\eta_{M 75\%}$	90.5 %
Motor efficiency 100% $\eta_{M 100\%}$	90.1 %
Protection class motor	IP55

Materials

Pump housing	Grey cast iron
Impeller	Stainless steel
Shaft	Stainless steel
Mechanical seal	BQ1EGG
Gasket material	EPDM

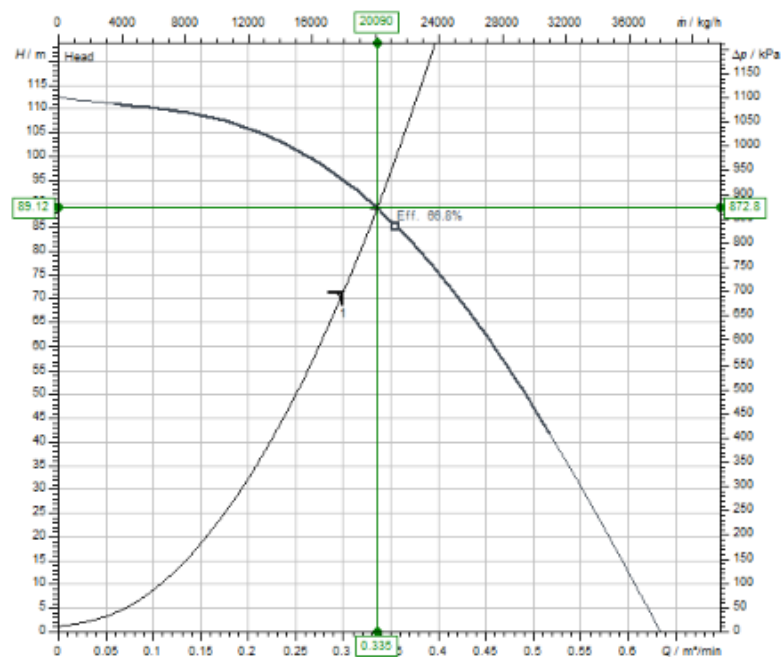
Installation dimensions

Pipe connection on the suction side DN_s	DN 50
Pipe connection on the discharge side DN_d	DN 50

Helix FIRST V 1606

wilo

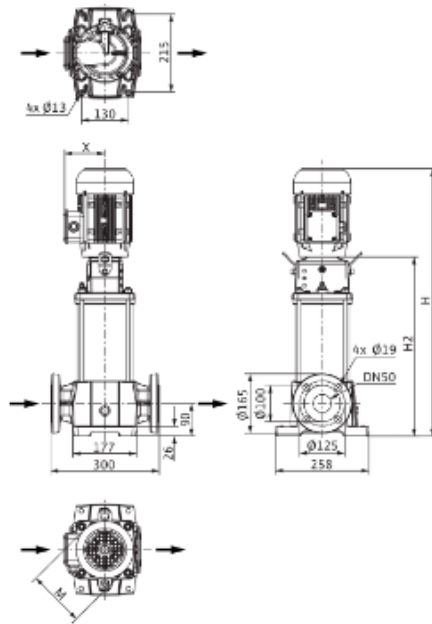
Pump curves



Fluid media	Water 100 %
Fluid temperature T	20.00 °C
Requested flow Q	0.30 m³/min
Delivery head (pressure unit)	71.49 m (700.00 kPa)
Delivered volume flow Q @ OP	0.34 m³/min
Delivery head (pressure unit) at the duty point p @ OP	89.12 m (872.69 kPa)
speed at duty point $n_{hydr.}$ @ OP	3569 1/min
Total electrical power consumption at the duty point $P1$ @ OP	8.14 kW
Total shaft power at the duty point $P2$ @ OP	7.34 kW
NPSH pump @ BP NPSH pump	2.08 m
Hydraulic efficiency at duty point $\eta_{hyd.}$ @ OP	66.66 %
Overall efficiency at the duty point $\eta_{tot.}$ @ OP	59.92 %

Dimensions and dimensions drawings

Helix FIRST V 16, PN 25



Helix FIRST V 1606



Dimensions <i>H</i>	1174 mm
Dimensions <i>H1</i>	152 mm
Dimensions <i>H2</i>	790 mm
Dimensions <i>L1</i>	150 mm
Dimensions <i>L2</i>	150 mm
Dimensions <i>L4</i>	130 mm
Dimensions <i>L5</i>	187 mm
Dimensions <i>L7</i>	0 mm
Dimensions <i>H3</i>	90 mm
Dimensions <i>H4</i>	850 mm
Dimensions <i>H5</i>	20 mm
Dimensions <i>H6</i>	429 mm
Dimensions <i>H7</i>	0 mm
Dimensions <i>L8</i>	183 mm
Dimensions <i>X</i>	196 mm